

KATHMANDU UNIVERSITY  
End Semester Examination [C]  
June/July 2024

Level : B.Tech.  
Year : III  
Time : 2 hrs. 30 mins.

07 JUL 2024

Course : MEEG 306  
Semester : I  
F.M. : 55

SECTION "B"

Attempt ALL questions. Assume suitable data if missing. Figures in the bracket refer to marks the question carries. Use of a data book is allowed.

1.

- a. Consider heat conduction equation as

$$\frac{d^2T}{dx^2} + \frac{d^2T}{dy^2} = \frac{1}{\alpha} \frac{\partial T}{\partial t}$$

[2]

- (i) Is heat transfer steady or transient?  
(ii) Is heat transfer one-, two-, or three-dimensional?  
(iii) Is there heat generation in the medium?  
(iv) Is the thermal conductivity of the medium constant or variable?

- b. An electric current is passed through a wire 1 mm in diameter and 10 cm long. The wire is submerged in liquid water at atmospheric pressure and the current is increased until the water boils. For this situation  $h = 5000 \text{ W/m}^2 \cdot ^\circ\text{C}$ . And the water temperature will be  $100^\circ\text{C}$ . How much electric power must be supplied to the wire to maintain the wire surface at  $114^\circ\text{C}$ ? [2]

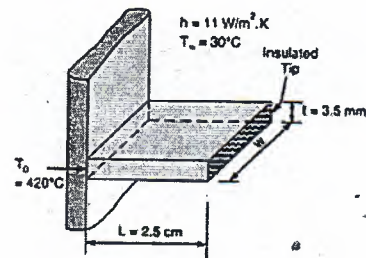
- c. Consider a large plate of thickness  $L$  and thermal conductivity  $k$  in which heat is generated uniformly at a rate of  $\dot{q}_{gen}$ . One side of the plate is insulated while the other side is exposed to an environment at  $T$  with a heat transfer coefficient of  $h$ . (a) Express the differential equation and the boundary conditions for steady one-dimensional heat conduction through the plate, (b) determine the variation of temperature in the plate, and (c) obtain relations for the temperatures on both surfaces and the maximum temperature rise in the plate in terms of given parameters. [6]

- d. What are various modes of heat transfer? Explain them briefly. [3]

2.

- a. Steam at  $T_{\infty,1} = 320^\circ\text{C}$  flows in a cast iron pipe ( $k = 80 \text{ W/m}\cdot\text{K}$ ) whose inner and outer diameters are  $D_1 = 5 \text{ cm}$  and  $D_2 = 5.5 \text{ cm}$ , respectively. The pipe is covered with 3-cm-thick glass wool insulation with  $k = 0.05 \text{ W/m}\cdot\text{K}$ . Heat is lost to the surroundings at  $T_{\infty,2} = 5^\circ\text{C}$  by natural convection and radiation, with a combined heat transfer coefficient of  $h_2 = 18 \text{ W/m}^2\cdot\text{K}$ . Taking the heat transfer coefficient inside the pipe to be  $h_1 = 60 \text{ W/m}^2\cdot\text{K}$ , determine the rate of heat loss from the steam per unit length of the pipe. Also, determine the temperature drops across the pipe shell and the insulation. [5]

- b. An aluminum alloy fin ( $k = 200 \text{ W/mK}$ ), 3.5 mm thick and 2.5 cm long protrudes from a wall. The base is at  $420^\circ\text{C}$  and the ambient air temperature is  $30^\circ\text{C}$ . The heat transfer coefficient may be taken as  $11 \text{ W/m}^2\cdot\text{K}$ . Find the heat loss and fin efficiency, if the heat loss from the fin tip is negligible. [3]



P.T.O.

- c. The steady-state temperature distribution in a one-dimensional wall of thermal conductivity 50 W/m K and thickness 50 mm is observed to be  $T(^{\circ}\text{C}) = a + bx^2$ , where  $a = 200^{\circ}\text{C}$ ,  $b = -2000^{\circ}\text{C}/\text{m}^2$ , and  $x$  is in meters. Determine the heat fluxes at the two wall faces. In what manner are these heat fluxes related to the heat generation rate? [1+1+1]

3.

- a. At a particular axial station, velocity and temperature profiles for laminar flow in a parallel plate channel have the form

$$u(y) = 0.75 \left[ 1 - (y/y_0)^2 \right]$$

$$T(y) = 5 + 95.66 (y/y_0)^2 - 47.83 (y/y_0)^4$$

Determine corresponding values of the mean velocity,  $u_m$ , and mean (or bulk) temperature,  $T_m$ . Plot the velocity and temperature distributions (by taking a few points). Do your values of  $u_m$  and  $T_m$  appear reasonable? [6]

- b. Consider a 0.6-m  $\times$  0.6-m thin square plate in a room at 30 $^{\circ}\text{C}$ . One side of the plate is maintained at a temperature of 90  $^{\circ}\text{C}$ , while the other side is insulated. Determine the rate of heat transfer from the plate if the plate is vertical. [4]

- c. The non-dimensional fluid temperature profile near the surface of a convectively cooled flat plate is given by  $\frac{T_w - T}{T_w - T_{\infty}} = a + b \frac{y}{L} + c \left( \frac{y}{L} \right)^2$  where  $y$  is measured perpendicular to the plate,  $L$  is the length, and  $a$ ,  $b$ ,  $c$  are arbitrary constants.  $T_w$  and  $T_{\infty}$  are wall and ambient temperatures respectively. If the thermal conductivity of the fluid is  $k$  and wall heat flux is  $q''$ . Find the value of Nusselt number. [4]

4.

The counter flow heat exchanger has a heat transfer area  $A = 10\text{m}^2$  and a corresponding overall heat transfer coefficient  $U = 500 \text{ W}/\text{m}^2\text{K}$ . It is used to cool 1.5 kg/s of hot oil initially at 100 $^{\circ}\text{C}$ , by contact with 0.5 kg/s stream of cold water whose inlet temperature is 15 $^{\circ}\text{C}$ . The  $c_p$  values of oil and water are 2.25 kJ/kg. K and 4.18 kJ/kg. K, respectively. Find the heat transfer rate and outlet temperatures. [4]

5.

A cylindrical heating element with a diameter of 1 cm and length of 30 cm is immersed horizontally in a pool of saturated water at atmospheric pressure. The cylindrical surface plated with copper. Calculate the heat flux  $q_w''$  and the total heat transfer rate from the cylinder to the water pool  $q_w$ , when the surface temperature is  $T_w = 108^{\circ}\text{C}$ . Calculate also the critical heat flux  $q''_{max}$ . Justify the values obtained with boiling regime curve. [4]

6.

- a. Two black parallel plates 0.5 m by 1.0 m are spaced 0.5 m apart. One plate is maintained at 1000 $^{\circ}\text{C}$  and other is maintained at 500 $^{\circ}\text{C}$ . What is net radiant heat exchange between the two plates? [3]

- b. For the Figure shown, determine  $F_{12}$ . [2]

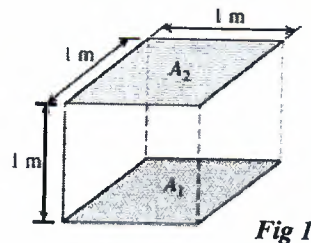


Fig 1

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- c. A spectral emissivity of an opaque surface at 800 K is approximated as shown in the figure below. Determine the average emissivity of the surface, maximum wavelength, and emissive power. [4]

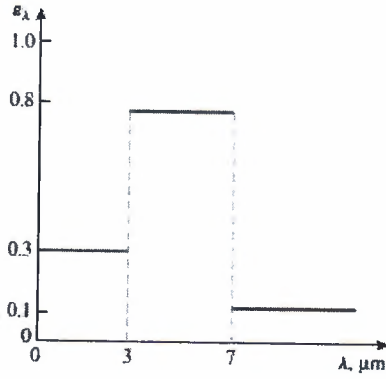
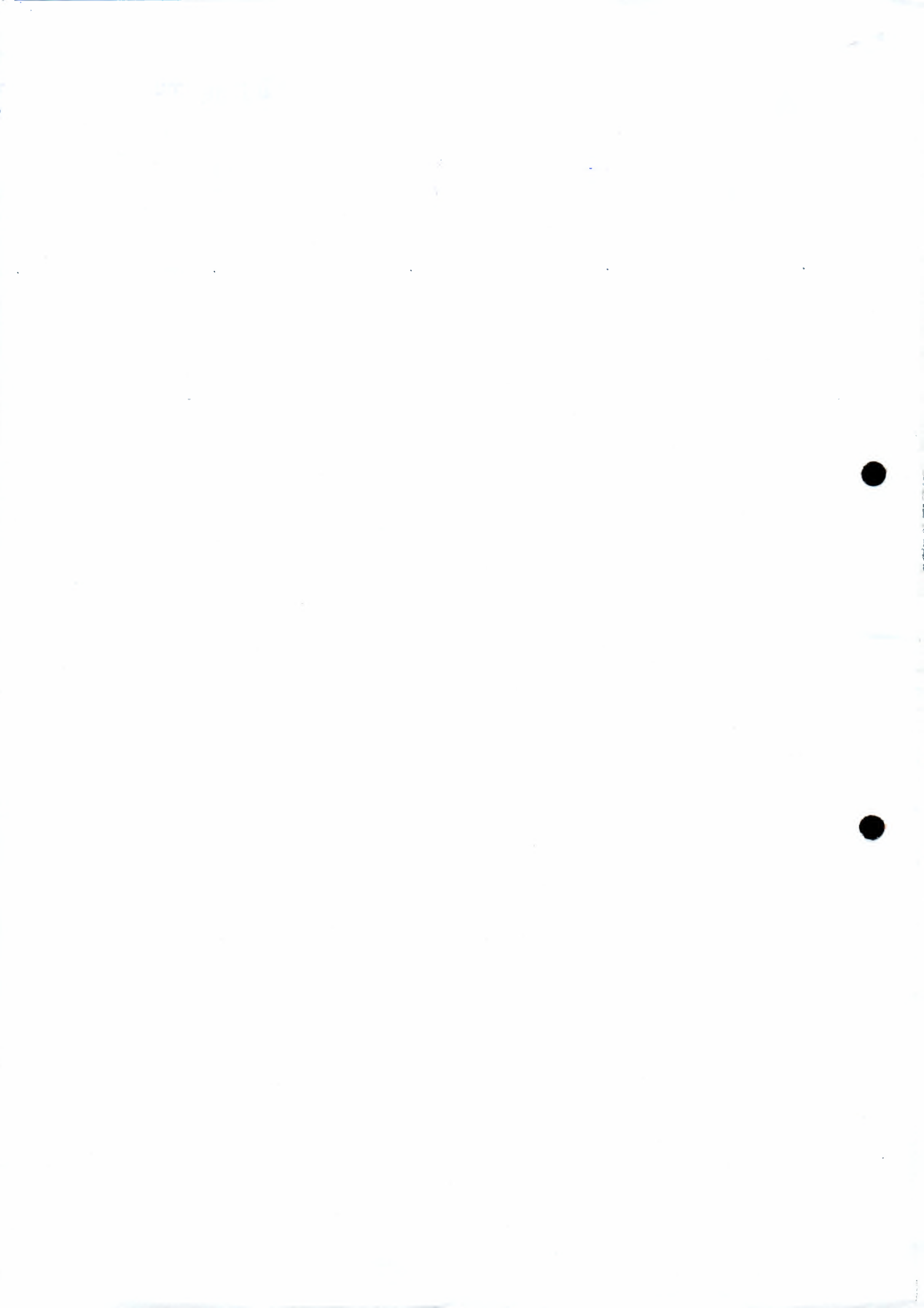


Fig 2



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F. M. : 20

Registration No.:

Date 07 : JUL 2024

SECTION "A"

[20Q. × 1 = 20 marks]

Choose and mark [X] the most appropriate option from each set of choices.

- The two end temperatures of a slab are maintained at 100°C and 50°C in presence of a heat source. The temperature profile obtained when the heat transfer takes place is  
 Linear       Parabolic       Exponential       Hyperbolic
- The unit of thermal conductivity  $k$  and heat transfer coefficient  $h$  are ..... respectively.  
  $\frac{W}{mK}$  and  $\frac{W}{m^2K}$         $\frac{W}{m^2K}$  and  $\frac{W}{mK}$         $\frac{W}{K}$  and  $\frac{W}{m^2K}$         $\frac{W}{mK}$  and  $\frac{W}{K}$
- Critical insulation radius for a cylinder is  
  $r_{o,c} = \frac{k}{h}$         $r_{o,c} = \frac{2k}{h}$         $r_{o,c} = \frac{h}{k}$         $r_{o,c} = \frac{2h}{k}$
- The temperature of the fin in case of long fin varies  
 Linearly       Hyperbolically       Exponentially       Parallel
- Consider steady one-dimensional heat conduction through a plane wall, a cylindrical shell, and a spherical shell of uniform thickness with constant thermophysical properties and no thermal energy generation. The geometry in which the variation of temperature in the direction of heat transfer will be linear  
 Plane wall       Cylindrical shell       Spherical shell       All of them
- Thermal diffusivity of a substance is  
 Inversely proportional to thermal conductivity  
 Direct proportional to thermal conductivity  
 Direct proportional to the square of thermal conductivity  
 Inversely proportional to the square of thermal conductivity
- The Nusselt number for the flow of water at 25 °C (velocity = 2 m/s) over a hot metallic sphere ( $d= 5$  cm) of 95 °C can be expressed as  
$$Nu = 2 + \left[ 0.4 \times \sqrt{Re} + 0.06 \times Re^{\frac{2}{3}} \right] Pr^{0.4} \left( \frac{\mu}{\mu_w} \right)^{0.25}$$

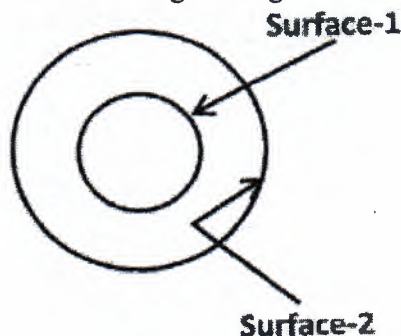
What will be the value of Nusselt number if the plate is placed in a stagnant pool of water at 45 °C? Where  $Pr = 7.86$  (at 45 °C);  $\left( \frac{\mu}{\mu_w} \right) = 1.15$  (at 45 °C)

 2       3.65       3.9       none of the options



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14. Air enters a duct at  $20^{\circ}\text{C}$  at a rate of  $0.08\text{ m}^3/\text{s}$ , and is heated to  $150^{\circ}\text{C}$  by steam condensing outside at  $200^{\circ}\text{C}$ . The error involved in the rate of heat transfer to the air due to using arithmetic mean temperature difference instead of logarithmic mean temperature difference is  
 5.4%                       8.1%                       10.6%                       13.3%
15. Internal force flows are said to be fully developed once the \_\_\_\_\_ at a cross section no longer changes in the direction of flow  
 Temperature distribution  
 Velocity distribution  
 The given condition is not enough to determine  
 Energy distribution
16. Air at  $20^{\circ}\text{C}$  flows over a 4-m long and 3-m wide surface of a plate whose temperature is  $80^{\circ}\text{C}$  with a velocity of 5 m/s. The length of the surface for which the flow remains laminar is (For air, use  $k = 0.02735\text{ W/m.K}$ ,  $Pr = 0.7228$ ,  $\nu = 1.798 \times 10^{-5}\text{ m}^2/\text{s}$ )  
 1.3m                       1.8 m                       2.2 m                       3.7 m
17. When boiling a saturated liquid, one must be careful while increasing the heat flux to avoid burnout. Burnout occurs when the boiling transitions from \_\_\_\_\_ boiling.  
 Convection to nucleate                       Convection to film  
 Film to nucleate                       Nucleate to film
18. An air handler is a large unmixed heat exchanger used for comfort control in large buildings. In one such application, chilled water ( $c_p = 4.2\text{ kJ/kg.K}$ ) enters an air handler at  $5^{\circ}\text{C}$  and leaves at  $12^{\circ}\text{C}$  with a flow rate of 1000 kg/hr. This cold water cools air ( $c_p = 1.0\text{ kJ/kg.K}$ ) from  $25^{\circ}\text{C}$  to  $15^{\circ}\text{C}$ . The rate of heat transfer between the two streams is  
 8.2 kW                       10.2 kW                       12.2 kW                       12.8 kW
19. The wavelength at which the blackbody emissive power reaches its maximum value at 300 K is  
 5.1  $\mu\text{m}$                        9.7  $\mu\text{m}$                        13.8  $\mu\text{m}$                        14.6  $\mu\text{m}$
20. A hollow enclosure is formed between two infinitely long concentric cylinders of radii 1m and 2m, respectively. Radiative heat exchange takes place between the inner surface of the larger cylinder (surface-2) and the outer surface of the smaller cylinder (surface-1). The radiating surfaces are diffuse, and the medium in the enclosure is non-participating. The fraction of the thermal radiation leaving the larger surface and striking itself is \_\_\_\_\_



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